



An Algorithm based on Predicting the Interface in Phase Change Materials

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ABSTRACT

Phase change materials are substances that absorb and release thermal energy during the process of melting and freezing. This characteristic makes phase change material (PCM) a favourite choice to integrate it in buildings. Stefan problem including melting and solidification in PCM materials is an practical problem in many engineering processes. The position of the moving boundary, its velocity and the temperature distribution within the domain are important for these applications. Well known numerical techniques have difficulties with time-dependent boundary conditions. Therefore, fine mesh and small time steps are needed to obtain accurate solutions. There are two main approaches to solve the Stefan problem: front-tacking and variable grid method. The most existing methods are not applicable to all situations and they cannot be easily implemented in two-dimensional or three-dimensional geometries and all boundary conditions. In this paper, we proposed an algorithm to solve one-dimensional Stefan problem in all kind of boundary condition; also it can be easily extended for 2D and 3D Stefan problems using finite difference method. For validation, the results are compared with exact solution of constant boundary condition. Afterward, periodic boundary condition is considered. The results showed significant relationship between numerical and exact solution, and the maximum error was approximately 0.4%.

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NOMENCLATURE

T	Temperature	k	Time index
T_0	Temperature at $x=0$	n	Position index
T_m	PCM melting temperature	*	Predicted value
T_b	Boundary temperature	N	Number of time steps
Δt	Time step	t	Time
Δx	Space step	x	Position
x^*	Position predicted value	A, D	matrices
S	Interface location between solid and liquid phases	Greek Symbols	
k	Conductivity (Wm/K)	ρ	Density (kg/m ³)
r	Latent heat (kJ/kg)	ε	Convergence criteria
a	Thermal diffusivity	α, β	Coefficient

1. INTRODUCTION

Increasing demand for thermal comfort can result in rising energy consumption in buildings and many other applications. PCM are substances that absorb and

release thermal energy during the process of melting and freezing. This characteristic makes PCM a suitable choice to integrate it in building walls. The use of PCM integrated into walls is a way to enhance the thermal storage capacity of buildings.

Stefan problems involving melting or solidification are important in many engineering applications [1-3].

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This problem is generally referred as phase-change or moving-boundary problem. The position of the moving boundary, its velocity and the temperature distribution within the domain are important for these applications.

Since the solid-liquid interface is time-dependent and must be determined as a part of the solution, the problems are highly nonlinear and become complicated except for a limited number of special cases. Therefore, these problems in the most cases are required to be solved numerically.

Due to difficulties in obtaining an analytical solution, various numerical techniques have been developed over last decades (including finite element, finite difference and integral methods) to solve moving boundary problem. Generally, in terms of accuracy and efficiency, the choice between various finite element, finite difference and integral methods for the solution of a particular Stefan problem is not always clear, due to their specific advantages and limitations.

A two-dimensional transient axi-symmetric model has been developed to study the effect of thermal and geometric parameters on cyclic heating and cooling modes of a phase-change thermal energy storage system by Adami [4]. The Gauss-Seidel iterative method with over-relaxation is used to solve the non-linear simultaneous difference equations [4].

A numerical simulation of refrigeration cycle with PCM heat exchanger carried out by Bakhshipour et al. [5] in cylindrical coordinate using finite volume approach. They investigated the effects of type of refrigerant, PCM heat exchanger length, PCM heat exchanger tube diameter, PCM thickness and mass flow rate of refrigerant. Costa et al. [6] have used the enthalpy formulation with a fully implicit finite difference method to analyse numerically the thermal performance of latent heat storage. The method takes into account both conduction and convection heat transfer in a one-dimensional model. The method used was validated by comparing the results with other analytical and numerical results found from the literature. The conclusion is that the method is useful for designing a thermal energy storage [6]. To obtain physical validation in PCM material, the numerical simulation has been carried out using enthalpy method and effective heat capacity method [7]. The numerical results compared with the experiment results achieved by thermocouples mounted inside the PCM.

Optimal homotopy asymptotic method (OHAM) has been used to solve one dimensional stefan problem by Rajeev et al. [8]. An approximate solution is obtained using the OHAM to find the solutions of temperature distribution in the domain $0 \leq x \leq s(t)$ and interface's tracking or location with the help of Taylor series.

Numerical techniques are specially known to have difficulties with time-dependent boundary conditions. Therefore, fine mesh and small time steps are needed to

obtain accurate solutions. Unfortunately, there is not enough research about the Stefan problem with time-dependent boundary conditions [9].

There are two main approaches to the solution of the Stefan problem [9]. One is the front-tracking method in which the position of the phase boundary will be tracked continuously. Alternatively, variable grid methods provide the way to track the phase front explicitly [10]. In variable grid methods, time step and space grid are variable.

For moving boundary problems, numerical methods have been compared in various studies [11].

In one algorithm proposed in literature [12], instead of front tracking, moving interface locations is preset and use these location coordinates as the grid points to find out the arrival time of moving interface. Applying this approach can help to avoid the difficulty in mesh generation. This algorithm encounters serious challenges in time-dependent boundary conditions because of presetting interface location and finding arrival interface time.

A mathematical model is proposed to simulate the coupled heat transfer equation and Stefan condition occurring in moving boundary problems like the solidification process in the continuous casting machines [13].

In this study, the finite difference approach and the boundary immobilization method has been selected to find the position of moving interface and the temperature distribution. Most of the proposed methods cannot be easily implemented in two / three-dimensional geometries and all boundary conditions. In this paper, we proposed an algorithm to solve one-dimensional Stefan problem in all kind of boundary conditions. This method can be easily extended for 2D and 3D Stefan problems. For this reason, this method presets arbitrary time steps and then it finds interface location.

To evaluate this new algorithm, both constant and time-dependent periodic boundary conditions have been considered, and the finite difference approach has been used in order to determine the temperature distribution and phase boundary during the process. Under constant boundary conditions, there is a significant relationship between the present numerical solution results and the exact solution one. To achieve qualitative results, grid independency is checked to find a suitable grid number with minimum error. The algorithm is also very straightforward and efficient for its finite difference formulation.

2. NUMERICAL ALGORITHM

In this section, a new algorithm will be numerically presented to solve Stefan problem in phase change material with arbitrary boundary conditions. The base of

this algorithm is predicting the boundary between the solid and liquid phase in each non-uniform time step. Also, at the same time steps can be selected. The discretization is based on finite difference schemes. The governing equation for one-dimensional heat transfer through a PCM is:

$$\frac{\partial^2 T}{\partial x^2} = \frac{1}{a} \frac{\partial T}{\partial t}$$

$$B.C. \begin{cases} T_1(x,t) = T_0 & \text{at } x=0 \text{ and } t > 0 \\ T_1(x,t) = T_m & \text{at } x=s(t) \text{ and } t > 0 \\ -k \frac{\partial T_1}{\partial x} = \rho r \frac{\partial s}{\partial t} & \text{at } x=s(t) \text{ and } t > 0 \end{cases} \quad (1)$$

where, T is temperature, t is time, T₀ is the temperature at x=0, T_m is the PCM melting temperature, ρ is the PCM density, k is thermal conductivity, a is thermal diffusivity, r is latent heat of PCM and s is the interface location between solid and liquid phases. The first boundary condition is constant temperature and third boundary condition is a nonlinear boundary condition coupled with governing equation. One of the main advantages of this algorithm is its ability to execute in every boundary condition like periodic and other types of boundary conditions. It is assumed that time steps are given as follows:

$$\begin{aligned} \Delta t_1 &= t_1 - t_0 = t^1 \\ \Delta t_2 &= t_2 - t_1 = t^2 \\ &\dots \\ \Delta t_k &= t_k - t_{k-1} = t^k \end{aligned} \quad (2)$$

Also, the interface in each time step defines as s_n^k that k is time index and n is the position index. If the PCM warms up at x=0 by T₀ temperature, and heat will be diffused through PCM and melting process will start. This process is illustrated in Figure 1.

Central scheme and Euler's first order scheme are used for discrete diffusion terms, and time derivative terms respectively. In the first time step, the length of melted zone can obtain discretizing the third boundary condition as Equation (3):

$$-k \frac{\partial T}{\partial x} = \rho r \frac{\partial s}{\partial t} \rightarrow -k \frac{T_1^1 - T_0^1}{\Delta x_1} = \rho r \frac{s_1^1 - s_1^0}{t^1} \quad (3)$$

$$T_1^1 = T_m, T_0^1 = T_0 \text{ and } \Delta x_1 = s_1^1 \text{ then } \rightarrow s_1^1 = \sqrt{t^1 \frac{k}{\rho r} (T_0 - T_m)}$$

The interface between liquid and solid phases at first time step will be easily predicted using only Stephen's boundary condition. But, in the next time steps to achieve this purpose, one dimensional heat transfer equation must be discretized based on Equation (4). As seen, a non uniform grid is used because of the non linear nature of Stephen's problem.

$$\frac{\partial^2 T}{\partial x^2} = \frac{\partial}{\partial x} \left(\frac{\partial T}{\partial x} \right) \text{ and } \left. \frac{\partial T}{\partial x} \right|_1 = \frac{T_{1+\frac{1}{2}}^1 - T_{1-\frac{1}{2}}^1}{(\Delta x_1 + \Delta x_2)/2}$$

$$\left. \frac{\partial^2 T}{\partial x^2} \right|_1 = \frac{\partial}{\partial x} \left(\frac{2 \left(T_{1+\frac{1}{2}}^1 - T_{1-\frac{1}{2}}^1 \right)}{\Delta x_1 + \Delta x_2} \right) = \frac{2}{\Delta x_1 + \Delta x_2} \left(\frac{T_m - T_1^2}{\Delta x_2} - \frac{T_1^2 - T_o}{\Delta x_1} \right) \quad (4)$$

$$\frac{\partial T}{\partial t} = \frac{T_1^2 - T_1^1}{\Delta t_2} = \frac{T_1^2 - T_m}{\Delta t_2}$$

Simplifying Equation (4), a relation like $\alpha T_1^2 = \beta$ will be obtained that α and β are as Equation (5).

$$\alpha = \frac{1}{\Delta x_2^*} + \frac{1}{\Delta x_1} + \frac{1}{a \Delta t_2}$$

$$\beta = \frac{T_1^1}{a \Delta t_2} + \frac{2}{\Delta x_1 + \Delta x_2^*} \left(\frac{T_m}{\Delta x_2^*} + \frac{T_o}{\Delta x_1} \right) \quad (5)$$

The * index indicates predicted values. With a similar procedure as mentioned in the first time step, the Δx₂ can be found as Equation (6).

$$\Delta x_2 = \sqrt{\Delta t_2 \frac{k}{\rho r} (T_1^2 - T_m)} \quad (6)$$

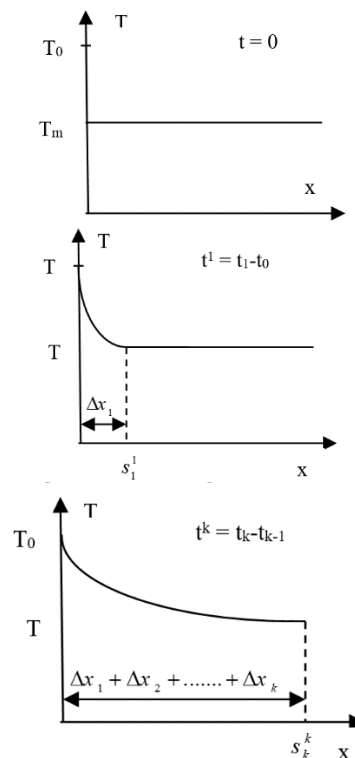


Figure 1. Schematic profile of temperature in a PCM material when melting process started

At the second time step, to start solution procedure, first a guess for Δx_2^* is necessary. After predicting Δx_2^* , α and β will be calculated explicitly using Equation (5). Afterward T_1^2 will be obtained from $\alpha T_1^2 = \beta$. Substituting T_1^2 in Equation (6) results in a new value for Δx_2 . The error criteria now can be found subtracting Δx_2^* from Δx_2 .

$$Error = |\Delta x_2 - \Delta x_2^*| < \epsilon \tag{7}$$

If error value was smaller than epsilon, convergence criteria are met and this step will be finished, otherwise, $\Delta x_2^* = \Delta x_2$ and this procedure repeats until the convergence criteria are met.

In the next steps, as the temperature of nodes couples together, the previous methods do not work correctly. Discretization of 1D heat equation results in Equation (8). Spatial derivative of Equation (1) is replaced with central difference approximation at n-1'th grid position. The n varies from 2 to k and k is current time step.

$$\begin{aligned} aT_{n-2}^k + bT_{n-1}^k + cT_n^k &= d \\ a &= \frac{-2}{(\Delta x_n + \Delta x_{n-1})\Delta x_{n-1}} \\ b &= \frac{2}{(\Delta x_n + \Delta x_{n-1})\Delta x_{n-1}} + \frac{2}{(\Delta x_n + \Delta x_{n-1})\Delta x_n} + \frac{1}{\Delta t^k} \\ c &= \frac{-2}{(\Delta x_n + \Delta x_{n-1})\Delta x_n} \\ d &= \frac{1}{\Delta t^k} T_{n-1}^{k-1} \end{aligned} \tag{8}$$

In the other hand, the above equations can be summarised as a system of the linear equation like $AT = D$. A and D are matrices which elements can be found in Equation (8) and T is the nodes temperature matrix. The convergence criteria at current time step can be found from third boundary condition at the interface of two phases, as expressed in Equation (9):

$$\Delta x_k^{new} = \sqrt{t^k \frac{k}{\rho r} (T_k^k - T_{k-1}^k)} \tag{9}$$

The algorithm of solving such a system can be listed as:

- A guess for Δx_k and solving $AT = D$ according to Thomas algorithm based on Equation (8).
- Finding Δx_k^{new} based on Equation (9).
- Checking convergence criteria $|\Delta x_k^{new} - \Delta x_k^{old}| \leq \epsilon$.
- If convergency happened, this algorithm finishes, else $\Delta x_k = \Delta x_k^{new}$ and this algorithm repeats from the first line.

The advantages of this algorithm are:

1. Grid generation has removed completely.
2. Compatible with different kinds of boundary

conditions like time dependent and etc.

3. The algorithm can be easily extended to solve Stephan problem in 2D and 3D.
4. This algorithm is flexible and easy to use.

3. RESULT

In this section, our algorithm results have been presented. Initially, to validate and verify the proposed algorithm, the analytical results of melting an aluminum wall under constant temperature are compared with numerical results. Then, two different boundary conditions (periodic boundary condition and constant boundary condition) are numerically investigated during a day. The PCM wall type is $\text{CaCl}_2 \cdot 6\text{H}_2\text{O}$.

In Figure 2, the length of melted zone is shown at different times and the analytic solution is presented to validate the numerical model. Melting process of aluminum is considered. The melting temperature of aluminum is $T_m=931$ K and boundary temperature is $T_b=1073$ K. The other physical properties of aluminum are given as follow: $r = 396$ kJ/kg, $\rho = 2380$ kg/m³, $k = 215$ Wm/K, $c = 1130.44$ J kg/K. The convergence criteria are $\epsilon = 1e-5$. As shown in Figure 2, numerical results are in good agreement with an exact solution that confirms the high accuracy of the numerical method. The maximum error happened in the first time and maximum error is approximately 0.4%.

In Figures 3 and 4 grid independency for two different boundary conditions (Figure 3 shows grid independency in $T_b=cte$ and Figure 4 presents it when periodic B.C. applied) are shown.

For this reason, a PCM wall with $\text{CaCl}_2 \cdot 6\text{H}_2\text{O}$ is selected. It is completely clear that in both states grid independency happens when the number of grid is $N = 96$. The temperature periodic function is as equation (10). Where T_b is boundary temperature, T_m presents melting temperature, N is the number of time steps and $n = 1, 2, \dots, N$. It should be mentioned that all next results obtained with $N = 96$ when grid independency happens.

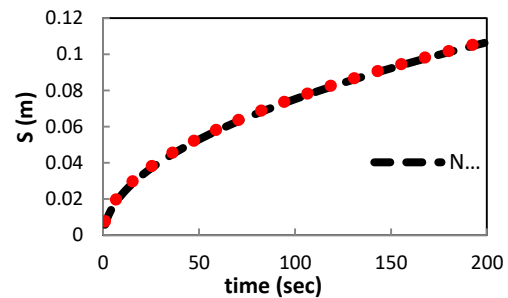


Figure 2. The comparison of numerical results with exact solution in melting process of aluminum under constant boundary condition

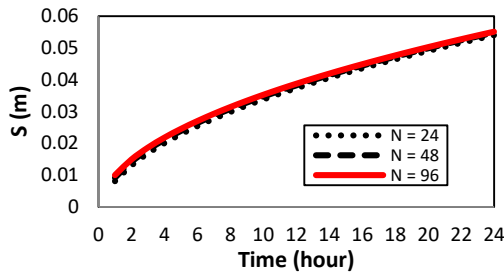


Figure 3. grid dependency for constant boundary condition

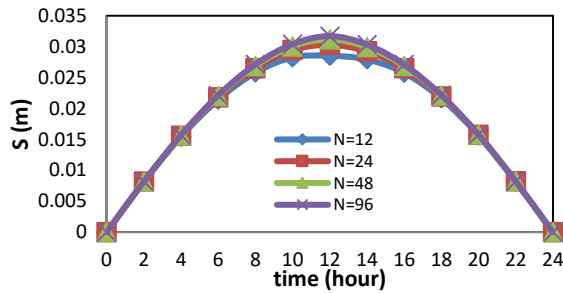


Figure 4. grid dependency for periodic boundary condition

$$T_b = T_m + T_0 \sin(2\pi n/N) \quad (10)$$

In Figure 5, temperature profile versus PCM melting zone length is shown in all day long ($t_{final} = 24$ h). The boundary temperature is $T_b = 314$ °C and PCM ($CaCl_2.6H_2O$) melting temperature is $T_m = 304$ °C. Other physical properties of $CaCl_2.6H_2O$ are: $k = 0.53$ W/mK, $r = 187$ kJ/kg, $\rho = 1530$ kg/m³ and $c_p = 2200$ kJ/kg.K. As shown in Figure 5 melting zone will be expanded when time increases. The maximum length of the melting zone is 5.51 cm and maximum rate of melting is in the first 6 hours where 2.64 cm of PCM melted. It can be easily understood that because of the nonlinear character of Stephan problem, PCM rate of melting is not the same in the same time steps. Therefore, using an algorithm based on non-uniform grid can result in more accurate results as this algorithm captures accurate melting zone length in PCM.

In Figure 6, temperature profile versus interface of the liquid and the solid region is shown. In this stage, periodic boundary condition is applied. As described in Equation (10) T_m is 304°C and T_0 is 10 °C. Also, N is the number of time steps and $n = 1, 2, \dots, N$. Simulation is done in all day long. In first twelve hours, melting process (as T_b is greater than T_m) and in the second section of the day freezing process (because T_m is greater than T_b) are applied based on periodic boundary condition. As illustrated in a Figure 6, the rate of melting in first six hours is greater than second six hours and interface length (between solid and liquid region) is 3.17 cm.

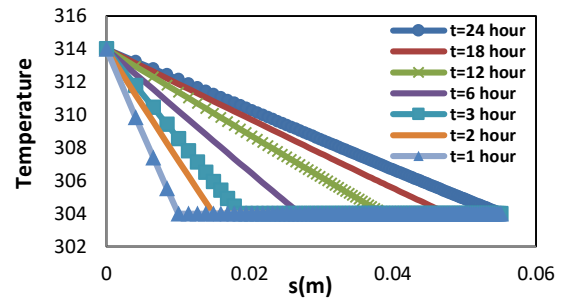


Figure 5. Temperature profile in melting process of $CaCl_2.6H_2O$ when boundary temperature is $T_0 = 314$ ° C

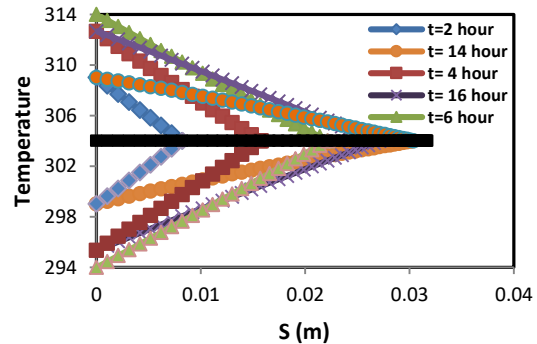


Figure 6. Temperature profile in $CaCl_2.6H_2O$ when boundary condition is periodic

In the other hand, optimum length for $CaCl_2.6H_2O$ is 3.17 cm when periodic boundary condition is applied.

But in the second twelve hours, PCM will be frozen. It is obvious from Figures 4 and 6 that the interface length is the same in times having the same distance from $t=12$ hours. It can be found that in periodic boundary conditions, melting and freezing process in PCM happen at the same interface exactly. Also, temperature profiles are completely symmetry in freezing and melting process. This confirms that the results are obtained under periodic boundary condition. The inverse of melting process, the freezing rate in the second six hours is greater than first six hours.

4. CONCLUSION

A new algorithm was proposed in this paper to solve one-dimensional Stephan problem based on finite difference method. This algorithm main features capturing moving interface in arbitrary time steps without mesh generation and it was easy to apply in different boundary conditions.

To evaluate this algorithm, two different boundary conditions (BC) have been considered. First, constant temperature BC was applied that numerical results had well coincident with analytical solution in an aluminum

wall. Then, time-dependent periodic BC was used in all day long and melting and solidification process in a PCM studied. The optimum PCM length captured approximately 3.17 cm for $\text{CaCl}_2 \cdot 6\text{H}_2\text{O}$ based on periodic boundary condition. Temperature profile versus interface location is linear in both boundary conditions. In melting and freezing process, the temperature profile was symmetry which confirms the validity of periodic boundary condition results. Also, Grid independency was checked to guaranty quality of numerical results.

The proposed algorithm can be easily developed for two-dimension and 3D applications. Algorithm accuracy was remarkable and maximum error was approximately 0.4% in the initial time step when constant temperature boundary condition was applied.

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An Algorithm based on Predicting the Interface in Phase Change Materials

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مواد تغییر فاز دهنده موادی هستند که انرژی گرمایی را در طی پروسه های ذوب شدن و انجماد آزاد می کنند. این مشخصه این مواد را به یک انتخاب مطلوب در کاربردهای ساختمانی تبدیل می کند. مساله استفان که شامل فرآیند ذوب و انجماد در مواد تغییر فاز دهنده می باشد، یک مساله کاربردی در بسیاری از کاربردهای مهندسی است. مکان مرز متحرک، سرعت آن و توزیع دما در دامنه حل در کاربردهای این مواد اهمیت دارد. تکنیکهای عددی شناخته شده مشکلاتی در مسایل مرز متحرک وابسته به زمان دارند. بنابراین چنین مسایلی برای داشتن حلی دقیق نیاز به شبکه ریز و گامهای زمانی کوتاه دارند. دو روش عمده برای حل مساله استفان موجود است. یک روش، روش ردیابی لبه و دیگری روش شبکه متغیر می باشد. اکثر روشهایی که در این حوزه بکار می روند جامع نیستند و به آسانی نمی توان از آنها در همه شرایط مرزی و مسایل دوبعدی یا سه بعدی استفاده کرد. در این مقاله یک الگوریتم برای حل مساله یک بعدی استفان که برای همه شروط مرزی قابل اجرا می باشد، ارائه شده است که به آسانی قابلیت تعمیم برای مسایل 2 بعدی و 3 بعدی را با استفاده از روش تفاضل محدود دارد. برای صحت سنجی، نتایج حاصله با حل دقیق در شرط مرزی دما ثابت مقایسه شده اند و سپس شرط مرزی پرودیک مورد توجه قرار گرفته است. نتایج حاکی از آن است که تطابق بسیار خوبی مابین نتایج حل عددی و حل دقیق وجود دارد و ماکزیمم خطا تقریباً 0.4٪ می باشد.

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