

## Comprehensive Simulation of Surface Texture for an End-Milling Process

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The analysis and simulation of the manufacturing process require extensive and complicated computations. Nowadays, computer resources and computational algorithms have reached the stage where they can model and simulate the problem efficiently. One of the important processes in manufacturing is machining. In this research, the end-milling process, which is one of the most complex and widespread processes in machining, is chosen. The most important parameters in end-milling are surface roughness and surface location errors. Comprehensive simulation software is developed to model the end-milling process, in order to anticipate the finishing parameters, such as surface roughness and errors. The proposed algorithm takes into account cutting conditions, such as feed, *doc*, *woc* and tool run out etc. In addition, the dynamic simulation module of the software can accurately model the flexible end-mill tool, the milling cutting forces and regeneration of the waviness effects, in order to construct a realistic surface texture model. The software can accurately determine the most commonly used index of surface roughness parameters, such as Ra, P.T.V. and surface errors.

### INTRODUCTION

End-milling operations are extensively used in the production of precise parts in industry. Milling process dynamics have been investigated in much research work. However, due to the complexity of the process, different fields of research are required. One area in the analysis and simulation of the milling process is in the developing of a comprehensive model, in order to anticipate surface quality, dimensional error and their trends in mass production. In various machining processes, the following steps are required to build up the model:

1. Analysis and modeling of the machining forces,
2. Modeling of the dynamic system of machine tools,
3. Modeling of the surface texture.

A comprehensive model for the accurate anticipation of surface errors requires a “precise” force model. The model must take account of phenomena such as, tool deflections, tool run-out, tool wear and machining

dynamics etc. In the most of previous research work, machining forces are obtained by using a mechanistic force model based on oblique cutting tests [1]. In this work, the orthogonal cutting theory and transforming method proposed by E. Budak et al. is used [2]. In contrast to previous methods, there is no need for an oblique cutting test for each tool shape. At first, a general machining database is provided using orthogonal cutting tests with various work materials and cutting tool materials. Using transforming relations and tool geometry, the cutting coefficients for oblique cutting are determined.

In general, one of the important sources of vibration in the milling process is the self-excited vibration. In this kind of vibration, internal sources will provide energy for the system. In some cases, the energy will exceed damping, thus vibrations sustain or even grow in amplitude. One of the important principles observed in machining dynamics is the regeneration of waviness, which affects chip thickness [3].

In the current research, a dynamic simulation model for machining dynamics is proposed. The surface is modeled using a 3D algorithm, which is an extension to available 2D methods [4]. The developed simulation model can anticipate surface roughness in any direction of interest on the machined surface.

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## SIMULATION OF CUTTING FORCES

Figures 1a and 1b illustrate the coordinate system and the end-milling geometry required for the model. Differential forces,  $df_t$  tangential,  $df_r$  radial and  $df_a$  axial are depicted, which act on the  $j$ th cutting edge for an ideal system (without any deflection). Values of the differential forces are obtained by the following (for tooth  $j$ ):

$$\begin{aligned} df_{tj}(\theta, z) &= [K_{te} + K_{tc}h_j(\theta, z)]dz, \\ df_{rj}(\theta, z) &= [K_{re} + K_{rc}h_j(\theta, z)]dz, \\ df_{aj}(\theta, z) &= [K_{ae} + K_{ac}h_j(\theta, z)]dz. \end{aligned} \quad (1)$$

In Equations 1 there are two separate terms, the first one is related to friction and ploughing forces and the second term is considered for the cutting action that takes place in the shear zone. Coefficients for the first phenomena are represented by  $K_{te}$ ,  $K_{re}$ , and  $K_{ae}$ , showing the amount of force per unit length of the cutting edge. Coefficients represented by  $K_{tc}$ ,  $K_{rc}$  and  $K_{ac}$  are considered for the cutting action, showing the amount of force per unit area of cutting edge.

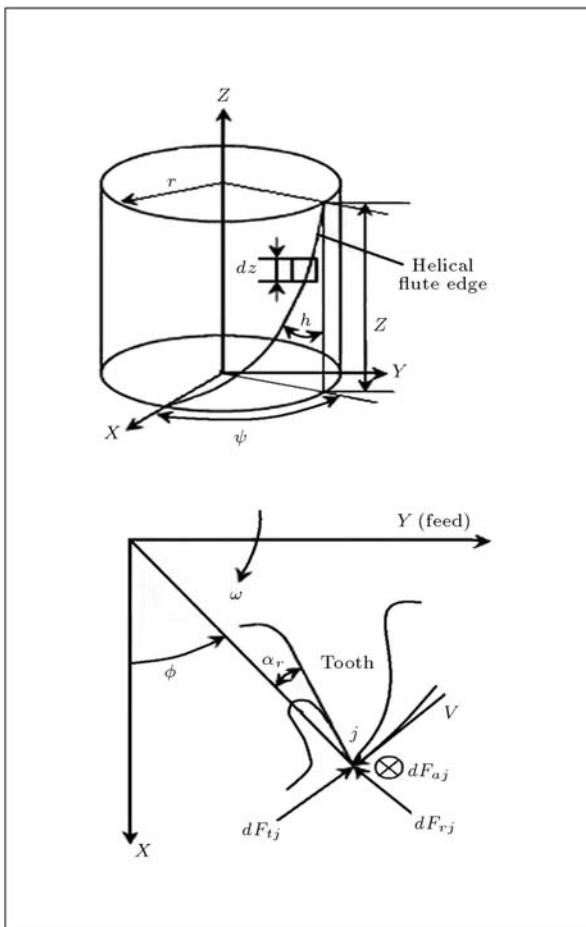


Figure 1. Coordinate system and end-milling geometry.

The simulation software, written in MATLAB V7.0, first computes cutting and edge force coefficients and then computes the cutting forces for each cutter angle [5]. Table 1 summarizes the computed coefficients for a four-flute HSS tool and the aluminium work-piece, 7075-T6, with a yield stress of 542 MPa. The computed cutting forces are depicted in Figure 2.

## REGENERATION OF WAVINESS

In general, the cutting edge of the tool is cutting a surface, which has been generated in the previous passes. If any vibrations exist between the tool and the work piece, the newly generated surface undulates, which results in the variation of chip thickness. Figure 3 illustrates the model used for the simulation of the milling process. The dynamic characteristics of the system in  $x$  and  $y$  directions are also depicted. Differential forces can be obtained by the following:

$$\begin{aligned} dF_{t_{n+1}} &= K_{tc}da(h_0 - z_n + z_{old1} + z_{old2} + z_{old3}), \\ dF_{r_{n+1}} &= K_{rc}da(h_0 - z_n + z_{old1} + z_{old2} + z_{old3}). \end{aligned} \quad (2)$$

Table 1. The computed coefficients for a four-flute HSS tool and an aluminium work-piece, 7075-T6, with a yield stress of 542 MPa. In this table,  $\alpha_n$  is the normal rake angle of the tool.

$\alpha_n$ (deg)	$K_{tc}$ [N/mm <sup>2</sup> ]	$K_{rc}$ [N/mm <sup>2</sup> ]	$K_{ac}$ [N/mm <sup>2</sup> ]
0	1642.3	609.5	512.2
12	1396.2	519.8	258.1
15	1356.5	504.4	215.3

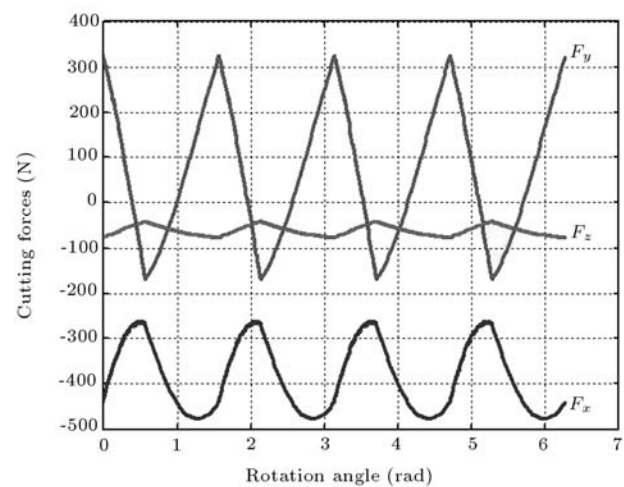
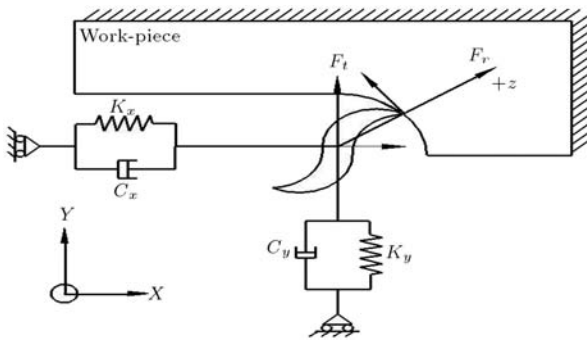


Figure 2. Simulated cutting forces for a four-flute HSS tool and an aluminium work-piece, 7075-T6, with yield stress of 542 MPa. Cutting conditions are  $d$  (tool diameter) = 10 mm,  $f_t$  (feed per tooth) = 0.05 mm/tooth,  $doc$  (depth of cut) = 10 mm.



**Figure 3.** Model used for simulation of the dynamic milling process.

Due to the fact that the axial stiffness of the tool and the spindle assembly is high and also that the axial force is smaller than the others, the axial vibrations are ignored. In the above relations,  $n$ ,  $h_0$ ,  $z_n$ ,  $z_{old1}$  to  $z_{old3}$  and  $da$  are, respectively, index of time increment, nominal chip thickness, tool deflection normal to the cut surface, tool deflection in previous passes and axial increment. In each instance, the differential cutting forces on the engaged cutting edges are computed and projected on an  $X$  and  $Y$  axis. Then, the total cutting forces are computed by integration of the differential forces over the engaged edges. Using Newton's second law, differential equations of motion are written and, then, using double Euler integration, the dynamic deflections are obtained as follows.

$$\begin{aligned} F_{x_{n+1}} &= m_x \ddot{x}_n + C_x \dot{x}_n + K_x x_n, \\ F_{y_{n+1}} &= m_y \ddot{y}_n + C_y \dot{y}_n + K_y y_n, \end{aligned} \quad (3)$$

where  $m_x$ ,  $m_y$ ,  $K_x$ ,  $K_y$ ,  $C_x$  and  $C_y$  are mass, stiffness and damping matrices for  $x$  and  $y$  directions, respectively. Input data for a four-flute end-mill with a helix angle of  $30^\circ$  are considered as follows [6,7]:

$$\begin{aligned} f_x = f_y &= 660 \text{ HZ}, & K_x = K_y &= 2 \times 10^7 \text{ N/m}, \\ M_x &= K_x / (2\pi f_x), & M_y &= K_y / (2\pi f_y), \\ \eta_x = \eta_y &= 0.05, & C_x &= \eta_x \times 2\sqrt{K_x M_x}, \\ C_y &= \eta_y \times 2\sqrt{K_y M_y}, & r &= 5 \text{ mm}, \\ n &= 1200 \text{ rpm}, & doc &= 10 \text{ mm}. \end{aligned} \quad (4)$$

In the above relations,  $f_x$ ,  $f_y$ ,  $\eta_x$  and  $\eta_y$  are the natural frequency and damping ratios for  $x$  and  $y$  directions, respectively. The tool radius, spindle speed and depth of cut are represented by  $r$ ,  $n$  and  $doc$ . The nominal chip thickness at angular position  $\theta$  is:

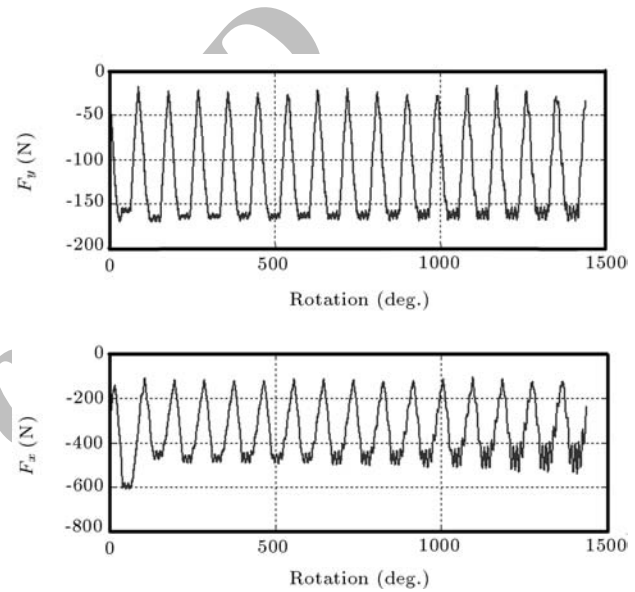
$$h_0 = ft \times \sin \theta. \quad (5)$$

Normal to cut deflection is:

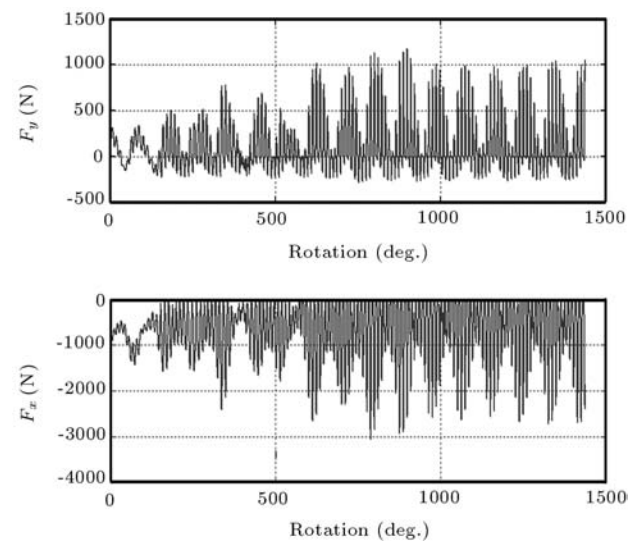
$$Z_n = x_n \sin \theta + y_n \cos \theta. \quad (6)$$

Using the proposed model, it is possible to include nonlinearities, such as jump-of-cut and chatter vibrations. Figure 4 shows the simulation results for an aluminium 7075-T6 work piece. Cutting conditions are  $d = 10$  mm,  $h$  (helix angle) =  $30^\circ$ ,  $ft = 0.1$  mm/tooth,  $woc$  (width of cut) =  $1.2$  mm,  $n = 1200$  rpm,  $doc$  (depth of cut) =  $10$  mm.

Dynamic milling forces for  $woc = 3$  mm are shown in Figure 5. The Fourier transform of the



**Figure 4.** Dynamic simulation of cutting forces for aluminium 7075-T6 work piece. Cutting conditions are  $d = 10$  mm,  $h = 30^\circ$ ,  $ft = 0.1$  mm/tooth,  $woc = 1.2$ .



**Figure 5.** Dynamic simulation of cutting forces for aluminium 7075-T6 work piece. Cutting conditions are  $d = 10$  mm,  $h = 30^\circ$ ,  $ft = 0.1$  mm/tooth,  $woc = 3$  mm.

milling forces are computed and depicted in Figures 6 and 7. Tooth passing is the dominant frequency in stable cutting. However, as inferred from Figures 6 and 7, the dominant frequency in unstable cutting is close to the natural frequency of the flexible mode shape, known as chatter frequency [6]. When  $woc = 3$  mm, milling is unstable and the vibration grows in amplitude gradually.

## DYNAMIC VIBRATIONS OF TOOL AND SPINDLE

Surface roughness prediction requires a comprehensive model in the time domain, which can simulate tool vibrations at locations of interest on the work piece. In the current research, milling vibrations are modeled by FEM using the CALFEM toolbox implemented in MATLABV7.0 software [8].

Figure 8 schematically shows the FEM model for the end-mill and spindle assembly, which is modeled as a step beam with two simple supports.

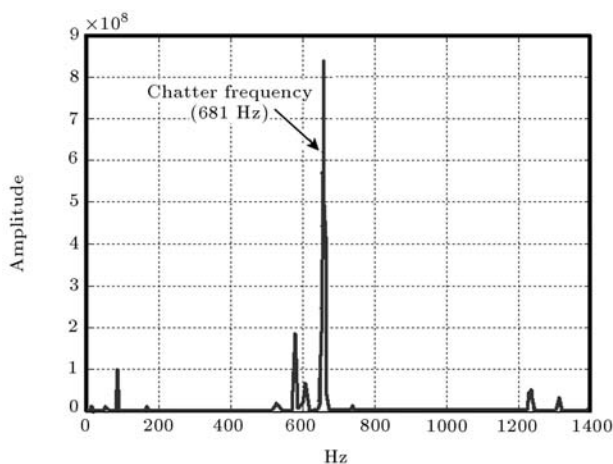


Figure 6. Fourier transform for  $F_x$ .

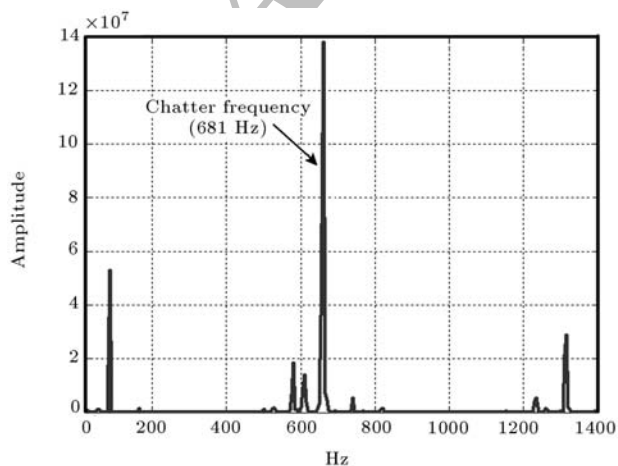


Figure 7. Fourier transform for  $F_y$ .

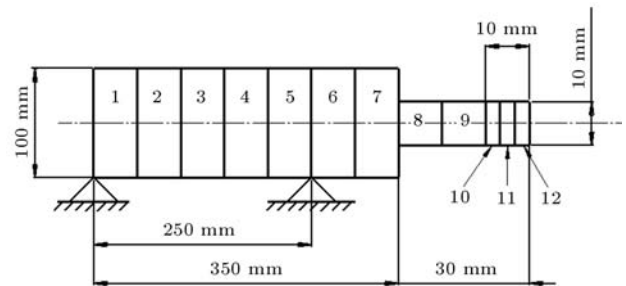


Figure 8. FEM model for the end-mill and spindle assembly.

The axial variation of the tool deflection is computed for the engaged portion of the tool, which is used by the surface location error algorithm. Therefore, finer elements in this portion are introduced.

Inputs for FE analysis are given in the following [9]:

modulus of elasticity of steel =  $207 \times 10^9$  (N/m<sup>2</sup>),

inner diameter of spindle axis = 0.02 m,

outer diameter of spindle axis = 0.1 m,

density of steel = 7800 (Kg/m<sup>3</sup>).

Figure 9 illustrates the developed flowchart of the FE analysis.

In the developed algorithm,  $Nr$ ,  $a$ ,  $da$ ,  $F_0$ ,  $T(N)$ ,  $X(N, j)$  and  $Y(N, j)$  are number of revolutions, axial depth of cut, axial increment, initial force, time and  $x$ - and  $y$ -deflections, respectively. Inputs to the algorithm are tool radius,  $woc$ ,  $doc$ , feed, cutting force constants and spindle speed. A function written in MATLAB V7.0 is used to calculate mass, stiffness and damping matrices. Using mesh and node positions, the mass, stiffness and damping of each element are computed and then assembled together to get overall matrices. In addition, boundary conditions and external cutting forces are applied. Differential equations of motion are integrated numerically by the Newmark method.

$$\begin{cases} M_x \ddot{X} + C_x \dot{X} + K_x X = F_X \\ M_y \ddot{Y} + C_y \dot{Y} + K_y Y = F_Y \end{cases} \quad (7)$$

For selected initial values of  $y = x = \dot{y} = \dot{x} = 0$ , the initial acceleration is expressed as follows:

$$\begin{aligned} \ddot{X} &= \frac{F - C_x \dot{x} - k_x x}{M_x}, \\ \ddot{Y} &= \frac{F - C_y \dot{y} - k_y y}{M_y}. \end{aligned} \quad (8)$$

The iteration loop starts from 1 and ends at  $a/da$  (where  $a$  is DOC and  $da$  is axial increment). Thus, the variation in axial deflection is obtained. Counter

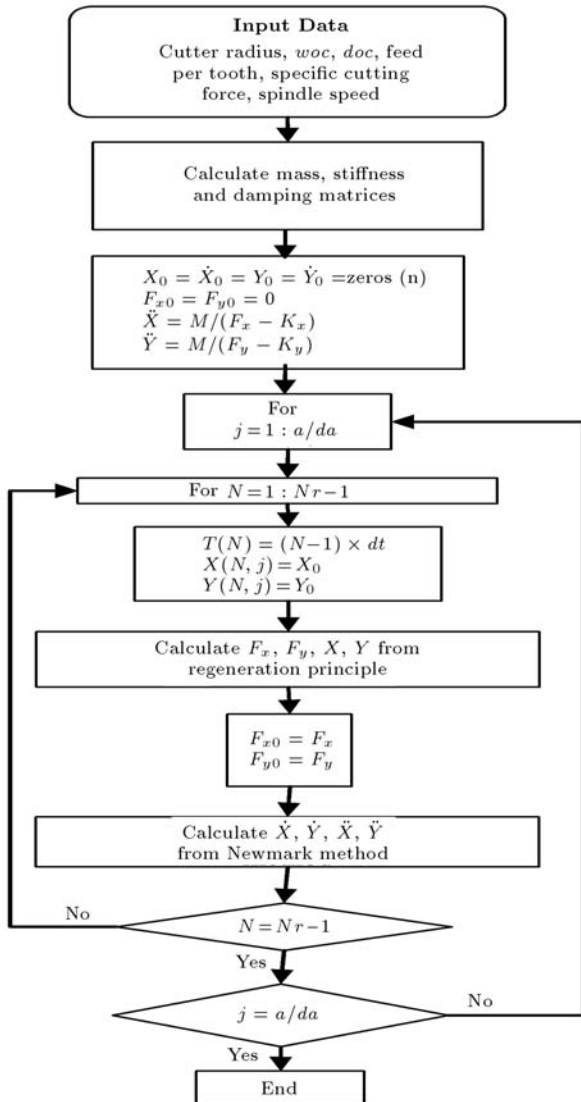


Figure 9. Developed flowchart of the FEM analysis.

$N$  varies from 1 to  $Nr - 1$ , which is considered for the time increments. After computing  $F_x$ ,  $F_y$ ,  $x$ , and  $y$  at current time increments, these values are used as initial values for the next step. Results of the milling tool in the time domain by FE analysis are shown in Figure 10. Vibrations of the tool center point are plotted at different axial levels.

In the next section, the amount of vibration is used for determining the cutting edge position and the surface generation method.

### TOOL PATH SIMULATION

The final machined surface is produced by subsequent cutting actions of end-mill cutting edges. Ideally, points on the cutting edge have trochoidal paths, which leave feed marks on the surface. Additionally, any tool deflection, vibration or geometric error results in

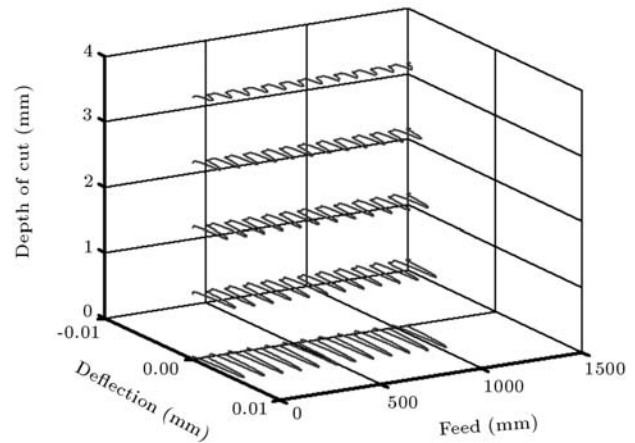


Figure 10. Results of time domain FE analysis of the milling tool for  $d = 12$  mm,  $f_t = 0.3$  mm/tooth,  $K_{tc} = 2000$  N/mm<sup>2</sup>,  $doc = 5$  mm,  $woc = 1.5$  mm.

undesirable deformation and deteriorates the required surface roughness on the work piece. The ideal position of the cutting edges, with respect to the tool cutter, is shown in Figure 11.

The XOY coordinate system is attached to the work piece and the original O is at the initial center position of the spindle. The trochoidal motion of a point on the cutting edge is computed by [9]:

$$X_{l/c}(i, z, t) = R \sin(\theta(t) - \psi(i) - \alpha(z)),$$

$$Y_{l/c}(i, z, t) = -R \cos(\theta(t) - \psi(i) - \alpha(z)), \quad (9)$$

where  $X_{l/c}$  and  $Y_{l/c}$  are coordinates of the edge, with respect to the tool center point. Also,  $i$ ,  $r$ ,  $\theta(t)$ ,  $\psi(i)$  and  $\alpha(Z)$  are tooth index, tool radius, angular position

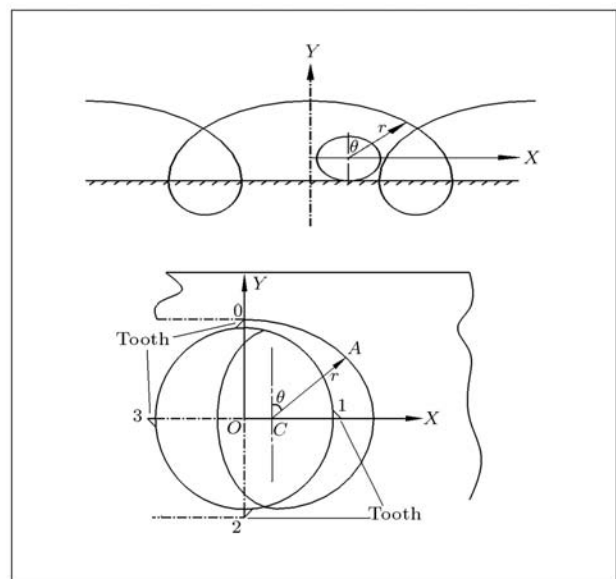


Figure 11. Trochoidal path of a point on the cutting edge.

of the lowest point of the reference cutting edge, pitch angle and phase lag of the point at  $z$  levels, respectively.

$\alpha(Z)$  is computed by:

$$\alpha(Z) = \frac{z \tanh h}{R}, \quad (10)$$

where  $h$  is tool helix angle.

On the other hand, displacement of the spindle center, with respect to the work piece, can be found by:

$$\begin{aligned} X_{s/w} &= f_r \times \theta(t)/(2\pi), \\ Y_{s/w} &= 0. \end{aligned} \quad (11)$$

If the runout error is not negligible, the location of the tool center, with respect to the spindle axis, will be computed by:

$$\begin{aligned} Y_{c/s}(t, z) &= -\rho \times \cos(\theta(t) + \lambda) + Y_{FEM}(t, z), \\ X_{c/s}(t, z) &= \rho \times \sin(\theta(t) + \lambda) + X_{FEM}(t, z), \end{aligned} \quad (12)$$

where  $\rho$  is runout offset and  $\lambda$  is its angular position (refer to Figure 12).

$X_{FEM}(t, z)$  and  $Y_{FEM}(t, z)$  are the dynamic tool deflections at time  $t$  and  $z$  level, which are calculated by FE analysis. Thus, instantaneous positions of a point on the cutting edge can be found by:

$$\begin{aligned} X_{l/w} &= X_{l/c} + X_{c/s} + X_{s/w}, \\ Y_{l/w} &= Y_{l/c} + Y_{c/s} + Y_{s/w}. \end{aligned} \quad (13)$$

Figure 13 shows the trochoidal motion of the cutting edges without considering vibrations and runout. In order to show the effects of the runout, its value is set to 1 mm, which is exaggerated intentionally. The other cutting conditions are kept constant. The simulation results are shown in Figure 14. Finally, Figure 15

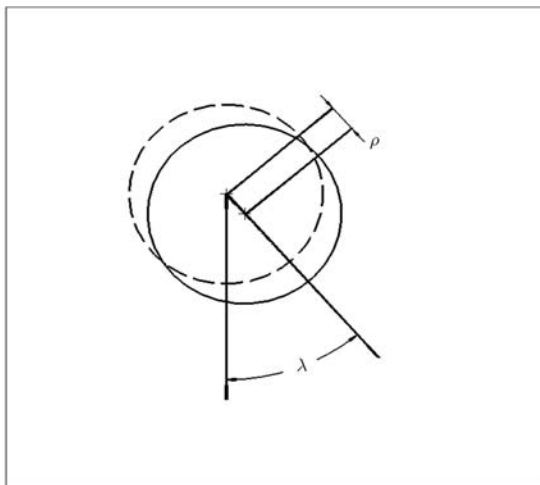


Figure 12. Runout parameters.

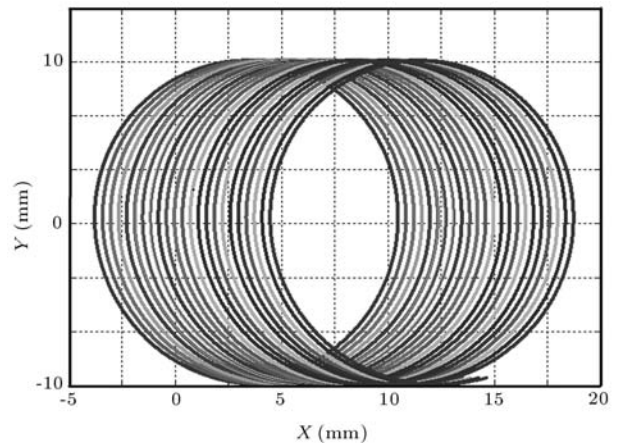


Figure 13. Trochoidal motions of the cutting edges without considering vibrations and runout ( $f_t = 1$  mm/tooth,  $d = 20$  mm).

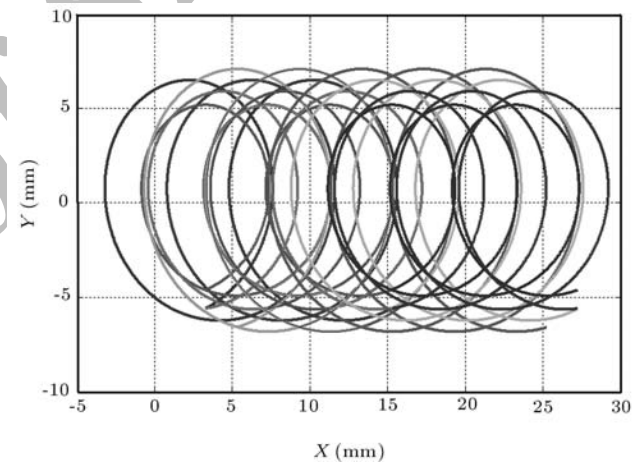


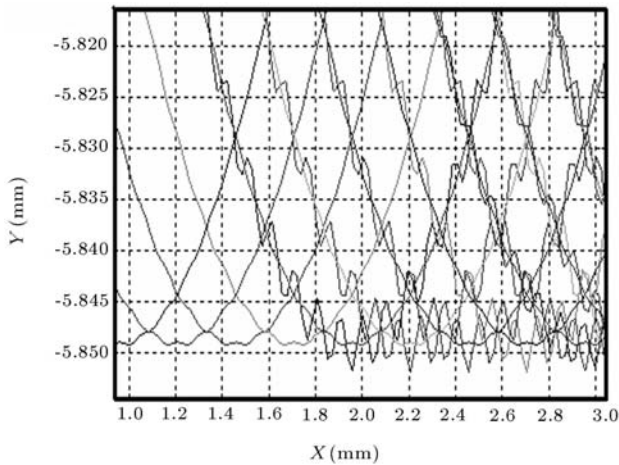
Figure 14. Trochoidal motions of the cutting edges with considering exaggerated runout ( $\rho = 1$  mm,  $\lambda = 45$ ).

shows the path of the cutting edge, including tool vibrations.

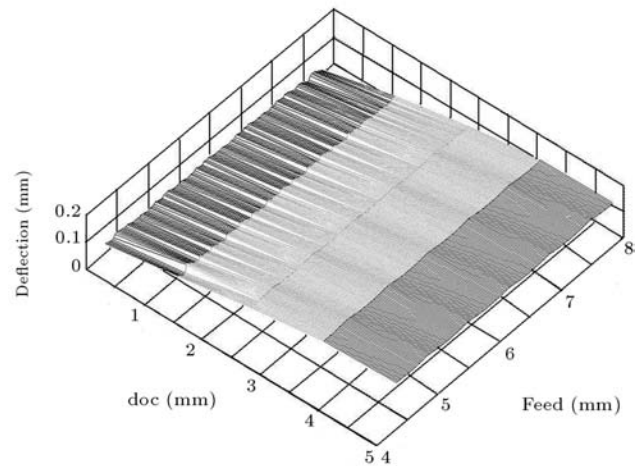
## SURFACE TEXTURE SIMULATION

Feed mark profiles can be produced by removing the points on the computed paths, which do not make contact with the newly generated surface. The  $z$ -level profiles are used in the mesh function to construct a 3D surface. Figures 16 to 18 illustrate 3D mesh surfaces for different cutting conditions. The cutting conditions used for the simulation of Figure 16, result in negligible vibrations. In order to show the runout effects, an exaggerated runout offset ( $\rho = 1$  mm) is chosen and the result of the simulation is shown in Figure 17. Extreme cutting conditions, which result in chatter vibrations, are used for the simulation of Figure 18.

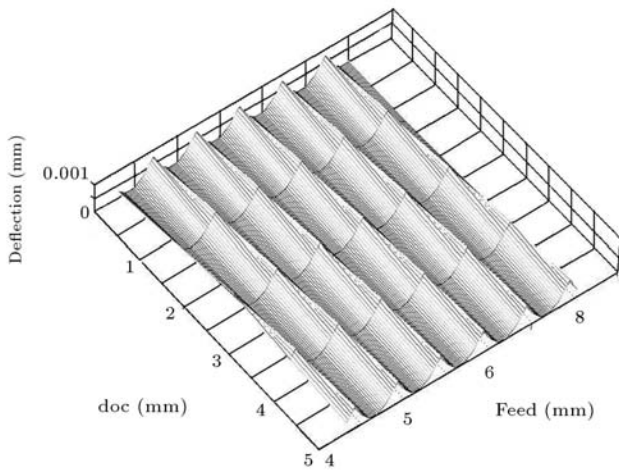
As shown in Figures 10 and 18, the amplitude of the vibrations at the tool tip is much higher. Table 2



**Figure 15.** Trochoidal motions of the cutting edges with considering vibrations ( $d = 12$  mm,  $f_t = 0.3$  mm/tooth).



**Figure 18.** Surface texture for  $d = 12$  mm,  $woc = 4.5$  mm,  $doc = 5$  mm,  $f_t = 0.6$  mm/tooth,  $K_{tc} = 1500$  N/mm<sup>2</sup>,  $\rho = 0.01$  mm ( $Ra = 14 \mu$ ).



**Figure 16.** Surface texture for  $d = 12$  mm,  $woc = 1$  mm,  $doc = 5$  mm,  $f_t = 0.1$  mm/tooth,  $K_{tc} = 1500$  N/mm<sup>2</sup> ( $Ra = 1.4 \mu$ ).

**Table 2.** Surface roughness computed at tool tip.

$d$ (mm)	$a$ (mm)	$WOC$ (mm)	$f_t$ (mm/tooth)	$Ra$ $\mu$
10	5	1.0	0.05	1.14
10	5	1.5	0.1	3.99
10	5	1.5	0.25	5.85
12	5	1.5	0.1	2.24
12	5	2	0.1	3.07
12	5	3	0.15	4.61

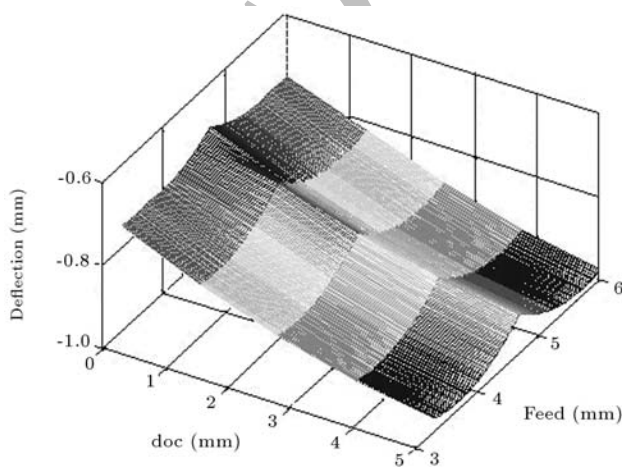
summarizes the  $Ra$  values of the surface roughness computed at the tool tip.

## CONCLUSIONS

In this research work, a comprehensive simulation model is developed for anticipating dynamic cutting forces and 3D surface texture for end-milling operations. The variation of instantaneous tool deflections are computed by FE analysis, using the CALFEM toolbox implemented in the MATLAB V7.0 environment. Variations of dynamic tool deflections are also computed along the axial direction for constructing the 3D surface texture model. The final model contains all the information required to compute the surface roughness parameter and surface location errors.

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**Figure 17.** Surface texture for  $d = 10$  mm,  $woc = 1$  mm,  $doc = 5$  mm,  $f_t = 0.1$  mm/tooth,  $K_{tc} = 1500$  N/mm<sup>2</sup>,  $\rho = 1$  mm ( $Ra = 3.5 \mu$ ).

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