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Predicting Combustion Process in Dual Fuel Engines by Incorporating Quasi-dimensional Multi-Zone and Detailed Chemical Kinetics Models

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Abstract

In the present work for prediction of combustion process in dual-fuel engines, a multi-zone combustion model for predicting combustion of pilot fuel and a chemical kinetic model for predicting gaseous fuel combustion are incorporated. Chemical kinetic model is consisting of 105 reactions with 31 species. Average pressure and mass average temperature, which are derived from two models, are considered as pressure and temperature of dual fuel engine. Predicted values of pressure and heat release rate from this modeling for dual fuel operation show good agreement with corresponding experimental data than the previous models.

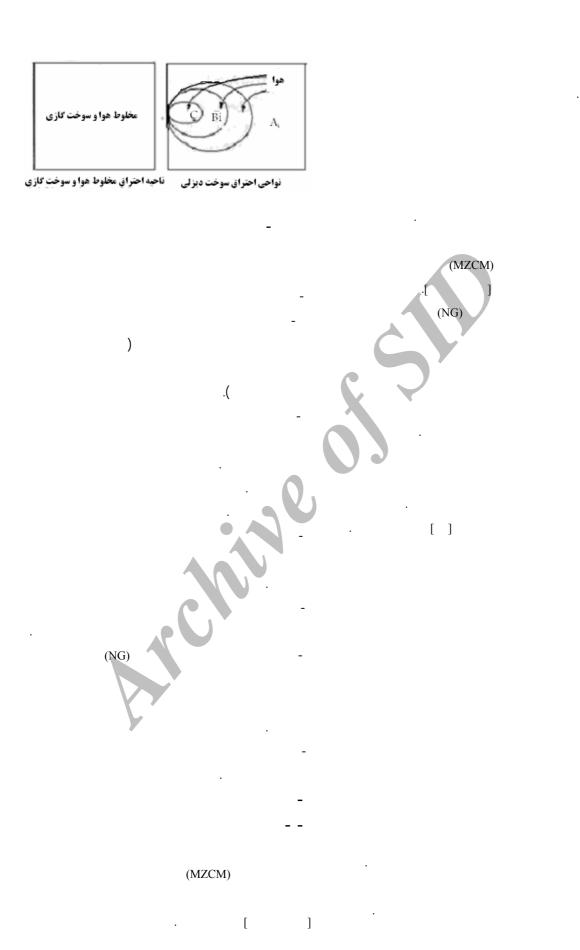
Key words: dual fuel, combustion modeling, chemical kinetics, pilot fuel, multi-zone model.

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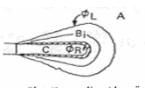
$$\begin{split} \frac{d(M_{bi}U_{bi})}{d\theta} &= -P\frac{dV_{bi}}{d\theta} + \frac{dQ_{bi}}{d\theta} + \frac{dM_{ebi}}{d\theta}H_{a} \qquad () \\ i &= 1, 2, 3, \dots, \quad (j-1) \\ \frac{d(M_{bj}U_{bj})}{d\theta} &= -P\frac{dV_{bj}}{d\theta} + \frac{dQ_{bj}}{d\theta} + \frac{dM_{mp}}{d\theta}H_{mp} + \frac{dM_{ebj}}{d\theta}H_{a} \qquad () \\ () & () & () \\ () & () & () \\ Q, V, M, U & . \end{split}$$

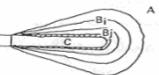
$$ebj, bj, bi, mp, ec, f, c$$

$$i$$

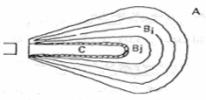
$$\vdots$$

$$T_{m} &= \frac{\sum M_{bi}T_{bi} + M_{a}T_{a} + M_{c}T_{c}}{\sum M_{bi} + M_{a} + M_{c}} \qquad ()$$





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 $\frac{dQ_{bi}}{dQ} = \left[\frac{M_{bi}T_{bi}}{\sum_{M}T_{m} + M_{m}T_{m} + M_{m}T_{m}} \right] \frac{dQ_{m}}{d\theta} \tag{)}$

.

$$P_i V_i = m R_i T_i \tag{)}$$

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 $\Sigma V_{bi} + V_a + V_c = V_{ch}(\theta)$ $\frac{d(M_c U_c)}{d\theta} = -P \frac{dV_c}{d\theta} + \frac{dQ_a}{d\theta} + \frac{dM_f}{d\theta} H_f + \frac{dM_{ec}}{d\theta} H_a - \frac{dM_{mp}}{d\theta} H_m$ () $\frac{d(M_a U_a)}{d\theta} = -P \frac{dV_a}{d\theta} + \frac{dQ_a}{d\theta} - \frac{dM_a}{d\theta} H_a \qquad ()$

: :

$$R_{jf} = k_{jf} \prod_{i=1}^{31} (\rho x_i)^{\alpha_{ijf}} \qquad \qquad () \qquad \frac{d(M_{ch}U_{ch})}{d\theta} = -P \frac{dV_{ch}}{d\theta} + \frac{dQ_m}{d\theta} + \frac{dM_f}{d\theta} H_f \qquad ()$$

$$R_{jb} = k_{jb} \prod_{i=1}^{31} (\rho x_i)^{\alpha_{jb}}$$

$$() \qquad \frac{dM_f}{d\theta} H_f$$

$$() () \qquad ()$$

$$\rho = \frac{M}{V(\theta)} \tag{)}$$

$$\rho \frac{dx_i}{dt} = \sum_{j=1}^{105} (\alpha_{ijf} - \alpha_{ijb})(R_{jf} - R_{jb})$$
(22H, C2H2, C2H3, C2H4, C2H5,C2H6,C3H8, NC3H7, iC3H7, C3H6, CH4, CH3, CH2, CH2CO, CH2O, CH3O, CH0, CH3CHO, CH3CO, CO, H2,H, CO2, C, HO2, H2O2, H2O, O, OH, N2, O2.

 $PV = MRT \tag{)}$

: k_{jf} j

:

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$\frac{dp}{d\theta}$	q									Q	$\dot{Q_p}$	
								MZCM)	()	()

- . II–Chemkin

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· [] OM-

/ kg/h / kg/h

 $q = \frac{\gamma_{av}}{\gamma_{av} - 1} p \frac{dV}{d\theta} + \frac{1}{\gamma_{av} - 1} V \frac{dp}{d\theta} + q_{wall} \qquad ()$

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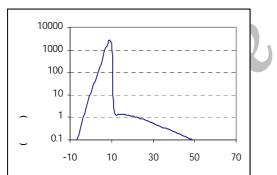
Methane	Ethane	Propane	i-Butane	n-Butane
1	1	01	1	1
neo- pentane	i- pentane	n- pentane	Hexanes	Carbon Dioxide
1	7	1	1	I
Nitrogen	Heptanes	octane		
1	V	1		

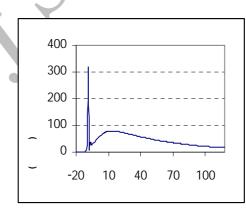
OM355

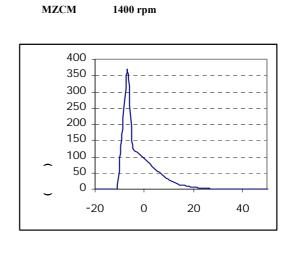
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rpm	
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rpm	
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400







[] 1400 rpm [] 1400 rpm

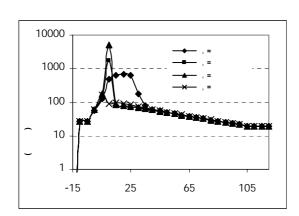
OM-P-θ)].] 80 10000 1000 100 10 -20 80 -120 -20 30 80 MZCM 1400 rpm MZCM 100000 10000 1000 100 10 -70 -20 30 80 -120 -20 30 80 1400 rpm [] []

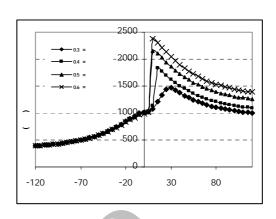
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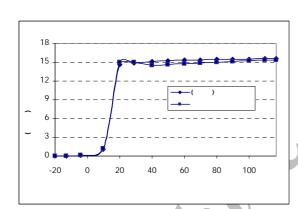
Kg/h

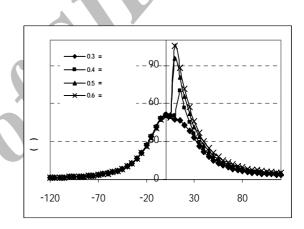
1400rpm

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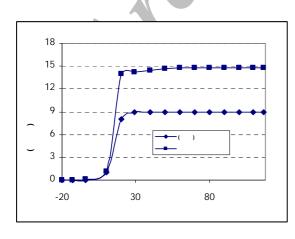








1400 rpm



 $\phi = 0.3$

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1400 rpm []

1- Accumulative Heat Release

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C_V
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mp
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     Callan.T.J. "Dual fuel operation of automotive
      diesel engine." M.Sc. Thesis, Department of
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      Mechanical Eng. University of Auckland, 1988.
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                                                                \boldsymbol{x}
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